

## CHAPTER 1 STEADY-STATE HEAT CONDUCTION

### 1.1 Heat Conduction in Multilayered Walls

The following example demonstrates calculation of the thermal resistance and temperature distribution within a wall assuming one-dimensional steady-state heat transfer. Note that in some cases different parts of the wall may have different layers, such as wood studs providing structural support. To determine a correct wall R-value in such cases, we need to calculate the correct value through each heat flow path and determine the overall R-value based on the relative area of each path.

#### Basic Equations

Heat flow  $q$  through a wall layer of thickness  $L$ , surface area  $A$  and thermal conductivity  $k$  is given by

$$q = \frac{T_{hot} - T_{cold}}{R} \quad \text{with} \quad R = \frac{L}{k \cdot A}$$

Heat flow  $q$  by convection or radiation described in approximate terms by a heat transfer coefficient  $h$  (convective, radiative, or combined):

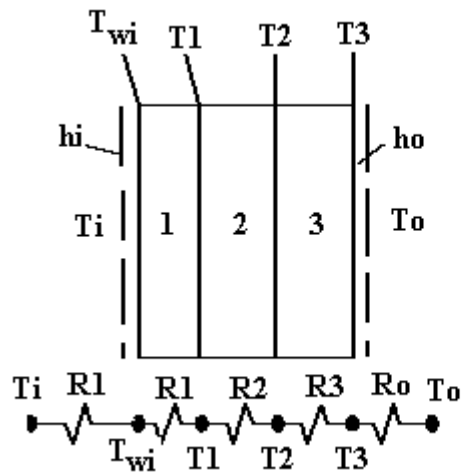
$$q = \frac{T_{hot} - T_{cold}}{R} \quad \text{with} \quad R = \frac{1}{h \cdot A}$$

Thermal resistance  $R_a$  through a path  $a$  is given by the sum of the resistances of the interior film, each wall layer and the exterior film. If a different path has thermal resistance  $R_b$  then the total wall resistance based on parallel heat flow paths is given by

$$R = \frac{A}{\frac{A_a}{R_a} + \frac{A_b}{R_b}} \quad \text{where} \quad A = A_a + A_b$$

In the example below, consider a wall with three layers:

1. Gypsum board
2. Insulation
3. Brick



Wall with  
three layers  
and interior and  
exterior films

Thermal network  
also shown

Input parameters:

$$h_i := 9 \frac{W}{m^2 \cdot \Delta^\circ C} \quad \text{interior heat transfer (film) coefficient}$$

$$h_o := 20 \frac{W}{m^2 \cdot \Delta^\circ C} \quad \text{exterior heat transfer coefficient}$$

$$A := 1.0 \, m^2 \quad \text{heat transfer area (surface)}$$

$$L_1 := 0.013 \, m \quad L_i = \text{thickness of layer } i$$

$$k_1 := 0.16 \frac{W}{m \cdot \Delta^\circ C} \quad k_i = \text{thermal conductivity of layer } i$$

$$L_2 := 0.05 \, m$$

$$k_2 := 0.025 \frac{W}{m \cdot \Delta^\circ C}$$

$$L_3 := 0.10 \text{ m}$$

$$k_3 := 1.5 \frac{\text{W}}{\text{m} \cdot \Delta^\circ\text{C}}$$

$$N := 3$$

N = number of layers  
(i denotes layer)

$$i := 1 \dots N$$

$$RI := \frac{1}{h_i \cdot A}$$

RI = interior film resistance

$$RO := \frac{1}{h_o \cdot A}$$

RO = exterior film resistance

$$R_i := \frac{L_i}{k_i \cdot A}$$

Ri = resistance of wall layer i

$$R_{tot} := RI + \sum_i R_i + RO$$

$$R_{tot} = 2.309 \frac{\Delta^\circ\text{C}}{\text{W}}$$

Rtot = total resistance of wall

Calculation of heat flow Q from inside (temperature TI) to outside (temperature TO):

$$TO := -20 \Delta^\circ\text{C}$$

$$TI := 20 \Delta^\circ\text{C}$$

$$m := 2 \dots N$$

$$Q := \frac{TI - TO}{R_{tot}}$$

$$T_{wi} := TI - Q \cdot RI$$

Twi = wall room side  
surface temperature

$$T_1 := T_{wi} - Q \cdot R_1$$

$$T_m := T_{m-1} - Q \cdot R_m$$

$$Q = 17.323 \text{ W}$$

$$T_{wi} = 18.075 \Delta^\circ\text{C}$$

$$i = \begin{bmatrix} 1 \\ 2 \\ 3 \end{bmatrix}$$

$$T_i = \begin{bmatrix} 16.668 \\ -17.979 \\ -19.134 \end{bmatrix} \Delta^\circ\text{C}$$