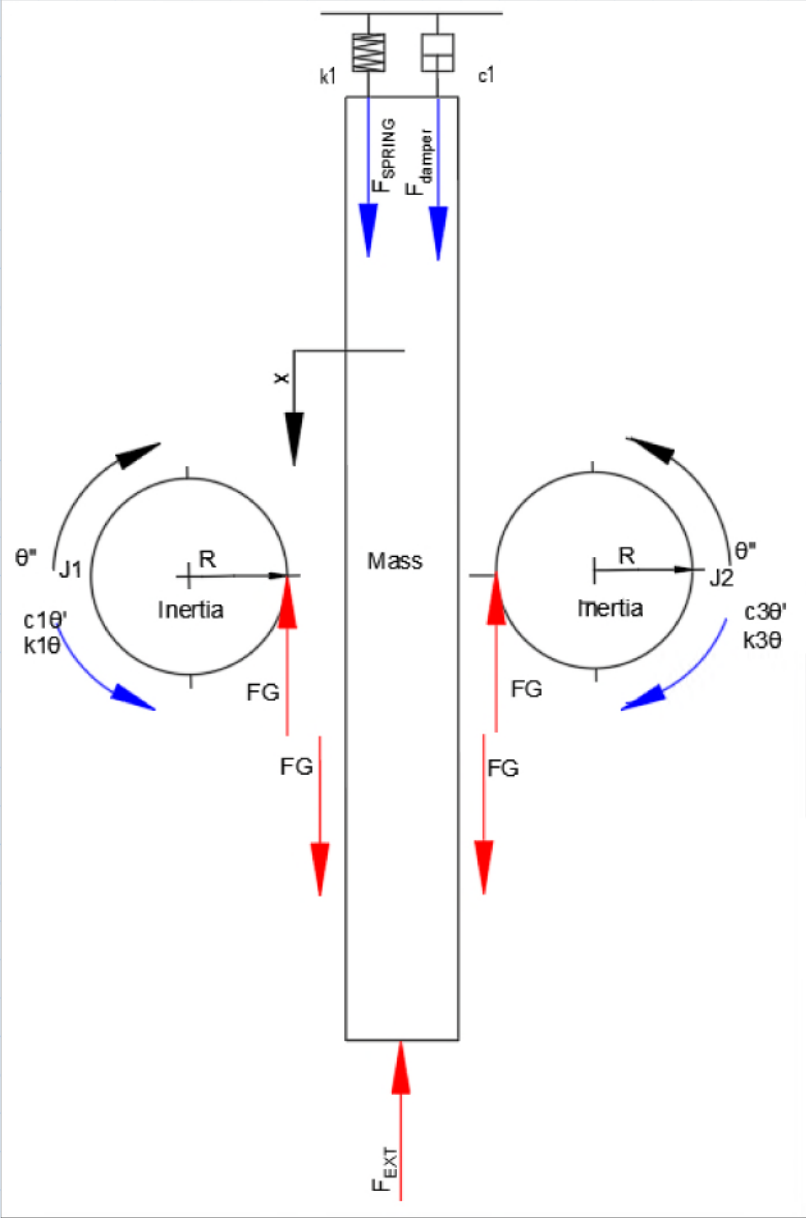
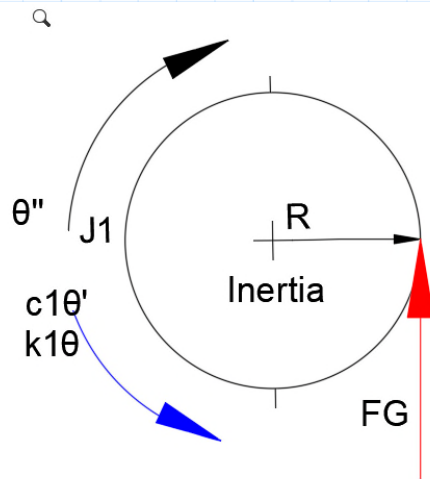


**Second-order differential equation to model the system behavior**



## Second-order differential equation Newton rotational law drive 1



For the rotation of the first motor (drive 1) according to Newton's laws with damping and stiffness:

$$\theta_1''(t) = \frac{-c_1 \cdot \theta_1'(t) - k_1 \cdot \theta_1(t) + k_2 \cdot \left( \frac{x_2(t)}{R_1} - \theta_1(t) \right) + c_2 \cdot \left( \frac{x_2'(t)}{R_2} - \theta_1'(t) \right) + T_{brake1} + \frac{m_{rack} \cdot g}{2} \cdot R_1}{J_1 + J_{1gear} \left( \frac{R_1}{R_2} \right)^2}$$

Mass of inertia 1 pinion  $J_1 := 0.01$

Mass of inertia 1 gearbox  $J_{1gear} := 0.02$

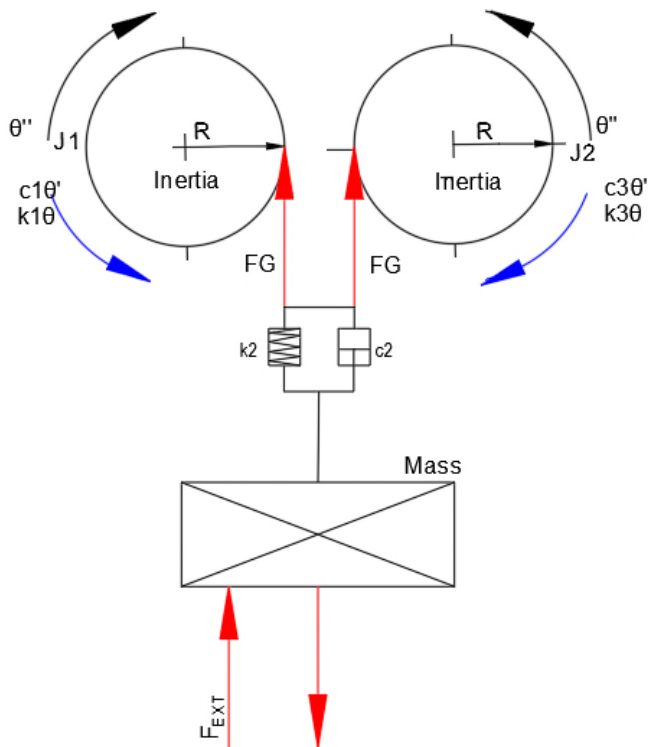
Pitch diameter pinion 1  $R_1 := 100$

Radial Damping Coefficient drive 1  $c_1 := 22590$

Radial Stiffness Coefficient drive 1  $k_1 := 1 \cdot 5.7 \cdot 10^7$

Brake  $T_{brake1} := 10$

## Second-order differential equation Newton rotational law rack



For the rack (mass-spring-damper system) according to Newton's laws with damping and stiffness:

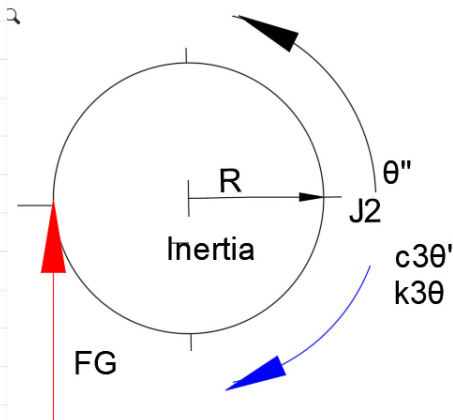
$$x_2''(t) = \frac{-k_2 \cdot (x_2(t) - \theta_1(t) \cdot R_1) - c_2 \cdot (x_2'(t) - \theta_1'(t) \cdot R_1) + k_2 \cdot (\theta_2(t) \cdot R_2 - x_2(t)) + c_2 \cdot (\theta_2'(t) \cdot R_2 - x_2'(t))}{m_{rack}}$$

Mass rack  $m_{rack} := 5$

Radial Damping Coefficient gear-pinion 1  $c_2 := 22590$

Radial Stiffness Coefficient gear-pinion 1  $k_2 := 1 \cdot 5.7 \cdot 10^7$

## Second-order differential equation Newton rotational law drive 2



For the rotation of the second motor (drive 2) according to Newton's laws with damping and stiffness:

$$\theta_2''(t) = \frac{-k_2 \cdot \left( \theta_3(t) - \frac{x_2}{R_2}(t) \right) - c_2 \cdot \left( \theta_3'(t) - \frac{x_2'}{R_2}(t) \right) + \frac{m_{rack} \cdot g}{2} \cdot R_2 - k_3 \cdot \theta_2(t) - c_3 \cdot \theta_2'(t)}{J_2 + J_{2gear} \left( \frac{R_1}{R_2} \right)^2}$$

Mass of inertia 2 pinion	$J_2 := 250$
Mass of inertia 2 gearbox	$J_{2gear} := 250$
Pitch diameter pinion 2	$R_2 := 100$
Radial Damping Coefficient damper 2	$c_3 := 22590$
Radial Stiffness Coefficient 2	$k_3 := 1.5.7 \cdot 10^7$
	$T_{brake2} := 20$
	$g_1 := 9.81$

Guess Values

$$\theta_1(0) = 0$$

$$z_2(0) = 0$$

$$\theta_3(0) = 0$$

$$\theta_1'(0) = 0$$

$$z_2'(0) = 0$$

$$\theta_3'(0) = 0$$

Constraints

$$\theta_1''(t) = \frac{-c_1 \cdot \theta_1'(t) - k_1 \cdot \theta_1(t) + k_2 \cdot \left( \frac{z_2(t)}{R_1} - \theta_1(t) \right) + c_2 \cdot \left( \frac{z_2'(t)}{R_2} - \theta_1'(t) \right)}{J_1}$$

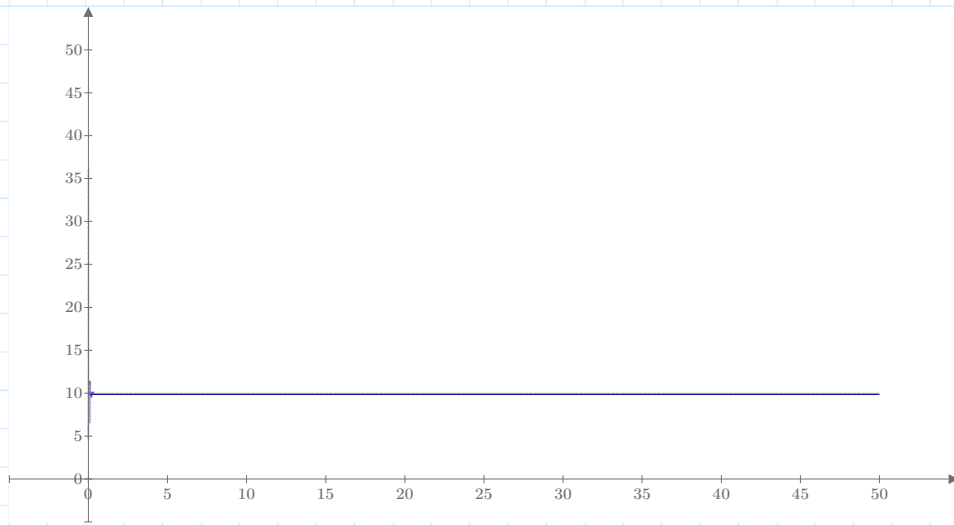
$$z_2''(t) = \frac{-k_2 \cdot (z_2(t) - \theta_1(t) \cdot R_1) - c_2 \cdot (z_2'(t) - \theta_1'(t) \cdot R_1) + k_2 \cdot (\theta_3(t) \cdot R_2 - z_2(t)) + c_2 \cdot (\theta_3'(t) \cdot R_2 - z_2'(t))}{m_{rack}}$$

$$\theta_3''(t) = \frac{-k_2 \cdot \left( \theta_3(t) - \frac{z_2(t)}{R_2} \right) - c_2 \cdot \left( \theta_3'(t) - \frac{z_2'(t)}{R_2} \right) + \frac{m_{rack} \cdot g_1 \cdot R_2}{2} - k_3 \cdot \theta_3(t) - c_3 \cdot \theta_3'(t)}{J_2}$$

Solver

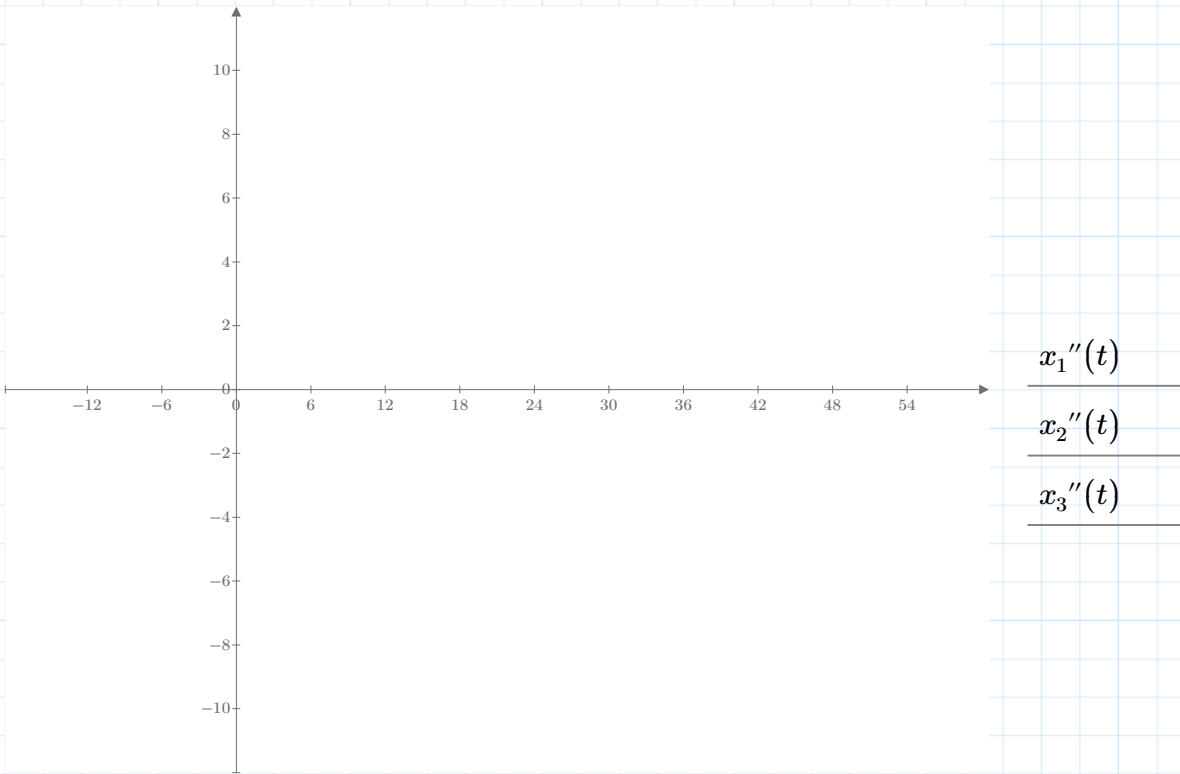
$$\begin{bmatrix} \theta_1 \\ z_2 \\ \theta_3 \end{bmatrix} := \text{odesolve} \left( \begin{bmatrix} \theta_1(t) \\ z_2(t) \\ \theta_3(t) \end{bmatrix}, 75 \right)$$

$t := 0, 0.0001 \dots 50$

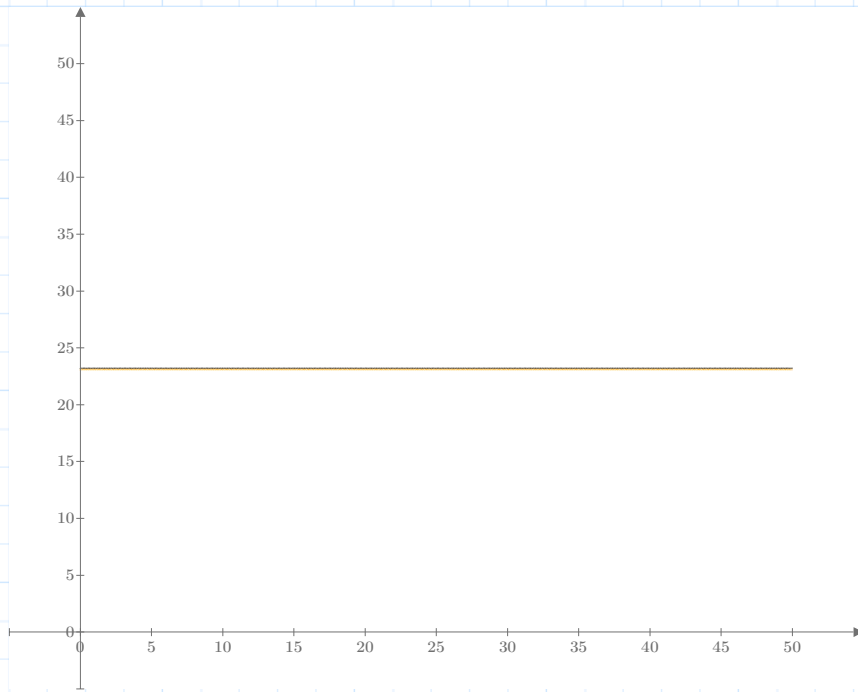


$\theta_1'(t)$

$t$



$t$



$$850000 \cdot g$$

$$m_3 \cdot \ddot{x}_3(t)$$

$t$