

$$T1 := 10$$

$$t1 := 0, \frac{T1}{1 \cdot 10^3} .. T1$$

Mass of body 1 $m_1 := 1$

Mass of body 2 $m_2 := 2$

Damping Coefficient $c_1 := 5$

Damping Coefficient $c_2 := 10$

Spring Constant k1 $k_1 := 100$

Spring Constant k2 $k_2 := 200$

External Force $F_1 := m_1 \cdot 9.81$

$$F_2 := m_2 \cdot 9.81$$

Given

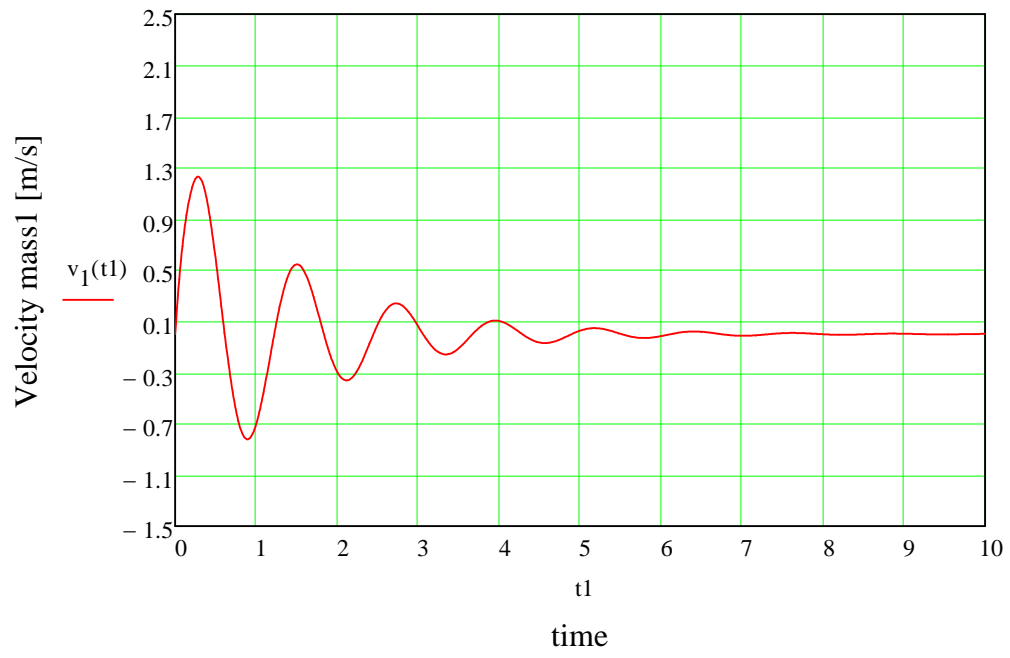
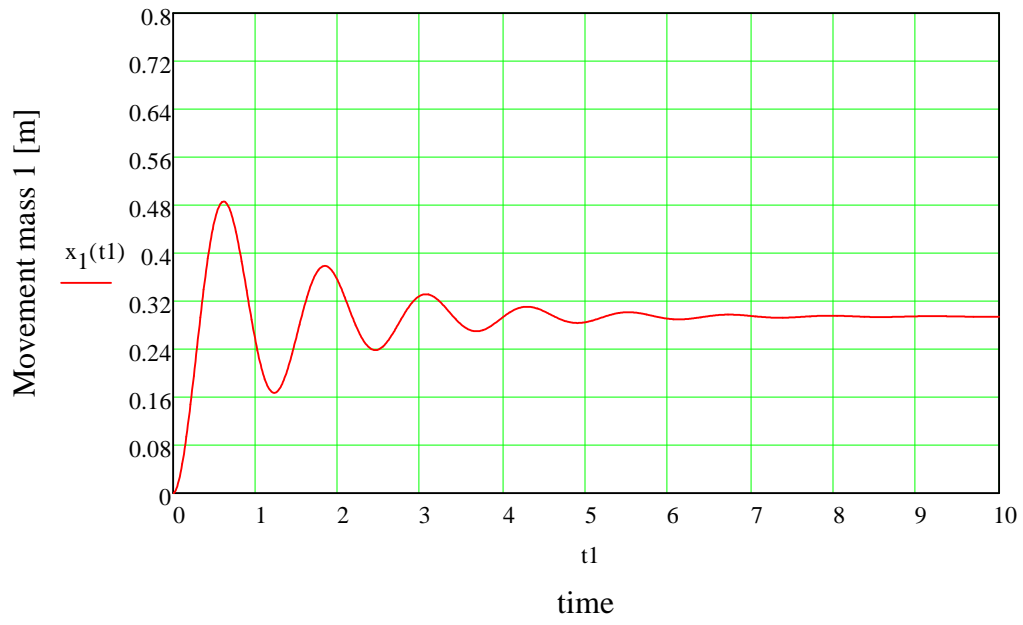
$$\frac{d}{du} y0(u) = y1(u) \qquad y0(0) = 0$$

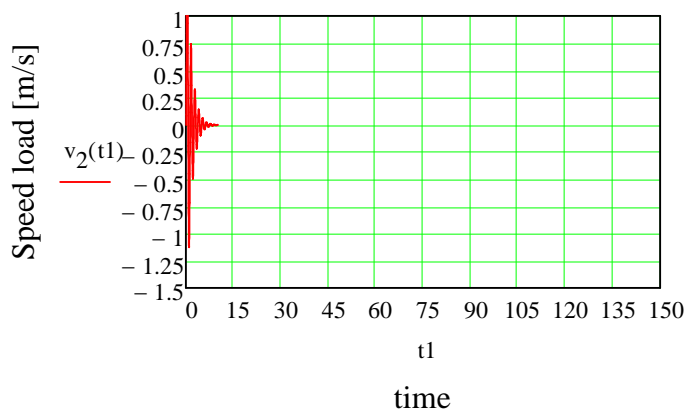
$$\frac{d}{du} y1(u) = -\frac{k_1}{m_1} \cdot y0(u) - \frac{c_1}{m_1} \cdot y1(u) + \frac{k_2}{m_1} \cdot (y2(u) - y0(u)) + \left(\frac{c_2}{m_1}\right) \cdot (y3(u) - y1(u)) + \frac{F_1}{m_1} \qquad y1(0) = 0$$

$$\frac{d}{du} y2(u) = y3(u) \qquad y2(0) = 0$$

$$\frac{d}{du} y3(u) = -\frac{c_2}{m_2} \cdot (y3(u) - y1(u)) - \left(\frac{k_2}{m_2}\right) \cdot (y2(u) - y0(u)) + \frac{F_2}{m_2} \qquad y3(0) = 0$$

$$\begin{pmatrix} x_1 \\ v_1 \\ x_2 \\ v_2 \end{pmatrix} := \text{Odesolve} \left[\begin{pmatrix} y0 \\ y1 \\ y2 \\ y3 \end{pmatrix}, u, T1, 10000 \right]$$



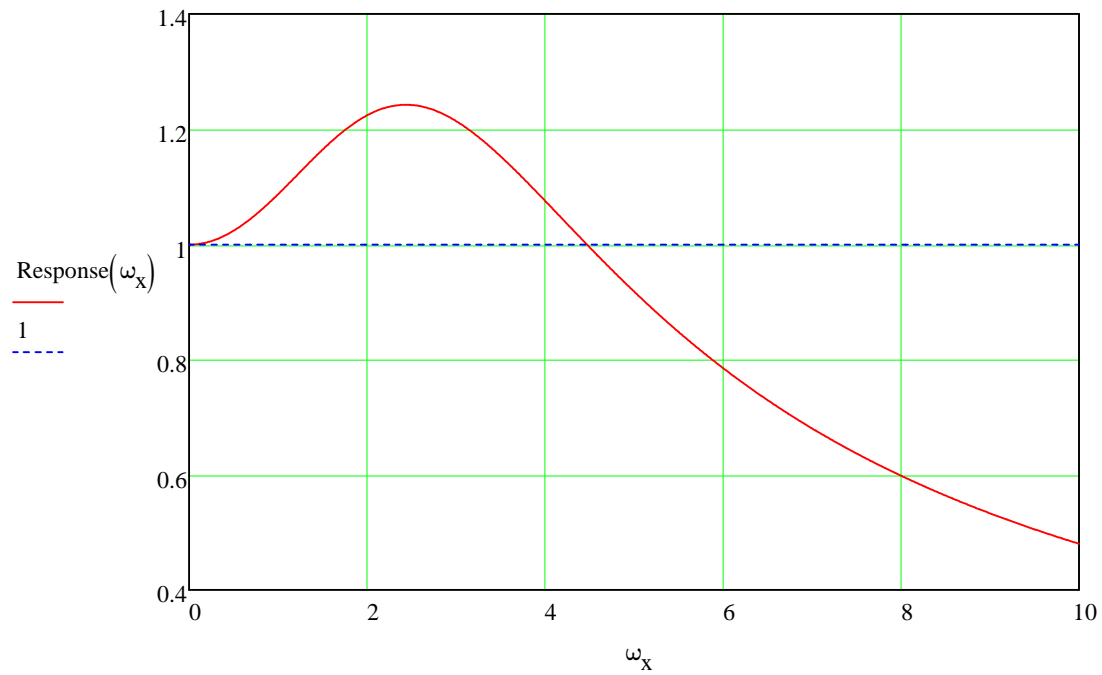


Mass module (incl. added mass)	$M_1 := 100000\text{kg}$	tons	
Drag damping on module	$c_{d1} := 500 \frac{\text{kN}\cdot\text{s}}{\text{m}}$	$\omega_1 := 1.047 \cdot \frac{1}{\text{s}}$	e
Compensator stiffness	$k_{c1} := 1000 \frac{\text{kN}}{\text{m}}$		
Compensator damping:	$c_{c1} := 20 \frac{\text{kN}\cdot\text{s}}{\text{m}}$		
Natural frequency	$\omega_n := \sqrt{\frac{k_{c1}}{M_1}}$	$\omega_n = 3.162 \frac{1}{\text{s}}$	
Periode natural frequency	$T_n := \frac{2\pi}{\omega_n}$	$T_n = 1.987\text{s}$	
Ratio between and angular frequency	$r(\omega) := \frac{\omega_1}{\omega_n}$	$r(\omega) = \blacksquare$	
Critical damping	$c_{\text{crit}} := 2 \cdot M_1 \cdot \omega_n$	$c_{\text{crit}} = 632.456 \cdot \frac{\text{kN}\cdot\text{s}}{\text{m}}$	
Damping ratio	$\beta := \frac{c_{c1} - c_{d1}}{c_{\text{crit}}}$	$\beta = -0.759$	

https://en.wikipedia.org/wiki/Damping_ratio

The system is under harmonic motion, and the response is defined as:

$$\text{Response}(\omega_x) := \frac{\sqrt{1 + \left(2 \cdot \beta \cdot \frac{\omega_x}{\omega_n}\right)^2}}{\sqrt{\left[1 - \left(\frac{\omega_x}{\omega_n}\right)^2\right]^2 + \left(2 \cdot \beta \cdot \frac{\omega_x}{\omega_n}\right)^2}}$$



The figure shows the response of the compensator. The peak corresponds to undamped natural frequency for the system. Peak is high above 1 and system should never be used for these frequencies. Dotted lines shows where the response becomes 1 for frequencies below this point

the compensator will not work.

